Amendments to the Specification

Please amend the paragraphs as follows:

Page 3, lines 6 to 14;

When fluid pressure is developed in the fluid pressure chamber, the inner lip is pressed into close contact with the outer periphery of the piston by the fluid pressure and the outer lip is pressed into close contact with a bottom wall of the concavity, in which the sealing member is received, by the fluid pressure, whereby the sealing member fluid-tightly ~~seal~~ seals between the outer periphery of the piston and the inner periphery of the cylinder bore.

Page 3, line 26 to page 4, line 14;

By the way, in the master cylinder disclosed in Japanese Unexamined Patent Publication No. 2003-194100, the thickness of the inner lip and the thickness of the outer lip are substantially equal. From this, it is considered that the inner lip and the outer lip both have the aforementioned sealing function and the pumping function. However, when the inner lip on which the piston slides ~~have~~ has the two functions, the inner lip must be somewhat thin in order to ensure the pumping function of the inner lip. When the inner lip is formed to be thin, however, a problem that the inner lip enters in the relief port, i.e. is caught by the piston, is caused. For this, in the master cylinder disclosed in Japanese Unexamined Patent Publication No. 2003-194100, the inner lip is provided with a tapered surface formed on a side thereof on which the piston slides, thereby preventing the inner lip from being caught by the piston.

Page 6, line 24 to page 7, line 18;

A master cylinder of the present invention comprises a

cylinder body having a cylinder bore, a piston which is slidably inserted into said cylinder bore and defines a fluid pressure chamber, a communication path which is formed in said cylinder body and communicates with a reservoir, a relief port which is formed in said piston, always communicates with said fluid pressure chamber, and allows communication between said communication path and said fluid pressure chamber, and a sealing member which is received in a concavity formed in the inner periphery of the cylinder bore of said cylinder body and into which the piston is slidably inserted so as to seal between the inner peripheral surface of said cylinder bore and the outer peripheral surface of said piston, wherein communication between said communication path and said relief port is allowed when the master cylinder is inoperative, and the communication between said communication path and said relief port is isolated by movement of said piston when the master cylinder is operative, and is characterized in that said sealing member is composed of a cup seal as claimed in any one of ~~claims 1 through 4~~ the first aspect through the forth aspect, and said base side fluid passage grooves communicate with said communication path.

Page 8, line 21 to page 9, line 11;

Further according to the cup seal of the present invention, the inner lip is formed to be thicker than the outer lip, that is, the inner lip into which the slidable member is slidably inserted is thick so that the inner lip is endowed only with the sealing function, while the outer lip is thin so as to allow the outer lip to be easily deflected and allow the hydraulic fluid to be easily sucked so that the outer lip is endowed both with the sealing function and the pumping function. Therefore, the cup seal is prevented from being caught by the slidable member, thereby improving the durability of the cup seal. According to the cup seal of the invention as claimed in ~~claim 4~~ the forth aspect, the

sealing function, the fluid supplying function, and fluid self-feeding function can be effectively exhibited with a simple structure without changing significantly the design of a conventional cup seal, just by forming grooves having a simple shape for allowing the flow of hydraulic fluid and designing the inner lip and the outer lip such that the inner lip is thicker than the outer lip.

Page 12, line 21 to page 13, line 8;

As shown in Figs. 1(a) and 1(b), the plunger-type master cylinder 1 is provided with ~~s~~ a first cylinder member 2. In the first cylinder member 2, a sleeve 3 is fitted fluid-tightly to the inner surface of the first cylinder member 2 by a sealing member 4 and a second cylinder member 5 is fitted fluid-tightly to the inner surface of the first cylinder member 2 by sealing members 6, 7 and is threadably fixed to the first cylinder member 2. Therefore, the sleeve 3 is caught between the first cylinder member 2 and the second cylinder member 5 in the axial direction and is thus fixed. The first and second cylinder members 2, 5 cooperate together to compose a cylinder body 32. An axial bore 2a of the first cylinder member 2, an axial bore 3a of the sleeve 3, and an axial bore 5a of the second cylinder member 5 cooperate together to form a cylinder bore 8.

Page 15, line 19 to page 16, line 17;

An annular first sealing member 27 composed of a cup seal is received in a concavity 26 between the sleeve 3 in which the primary piston 9 is disposed and the second cylinder member 5 so that the primary piston 9 is fluid-tightly and slidably inserted through the first sealing member 27. As shown in ~~Fig. 2(a)~~ Figs. 2(a) through 2(c), the annular first sealing member 27 comprises an annular base portion 27a which radially extends and through which

the primary piston 9 is slidably inserted, an annular inner lip 27b which extends axially from the inner side end of the base portion 27a and through which the primary piston 9 is slidably inserted, and an annular outer lip 27c which extends axially from the outer side end of the base portion 27a and is in contact with a bottom wall 26a of the concavity 26 such that the outer lip 27c can be spaced apart from the bottom wall 26a. By these components, the annular first sealing member 27 is formed to have a laterally-facing U-shaped section.

The outer lip 27c is formed to be relatively thin. The inner lip 27b has a thickness larger than that of the outer lip 27c and substantially equal to that of the base portion 27a. Therefore, the tip end portion of the outer lip 27c is easily deflected so as to allow easy suction of fluid. The length of the outer lip 27c in the axial direction is shorter than the length of the inner lip 27b so that the inner ~~lop~~ lip 27b has a portion $27b_1$ which is not overlapped with the outer ~~lop~~ lip 27c.

Page 26, lines 10 to 26;

By the spaces formed between the ends of the first and the second relief ports 23, 25 of the both pistons 9, 10 and the ends of the first and second sealing members 27, 28 in the inoperative positions, the ~~communication~~ communications between the first and second sealing members 27, 28 and the communication paths 22, 27 are allowed via the base side grooves, respectively, when the master cylinder 1 is inoperative. Therefore, the communication paths 22, 24 are prevented from being closed by the first and second sealing members 27, 28 being caught by the first and second communication paths 22, 24 due to fluid pressure developed in the first and second fluid pressure chambers 11, 12 of the master cylinder 1, for example, when the automatic braking is cancelled or an antilock braking control is cancelled. Therefore, the flow of

brake fluid is never blocked at the first and second communication paths 22, 24.